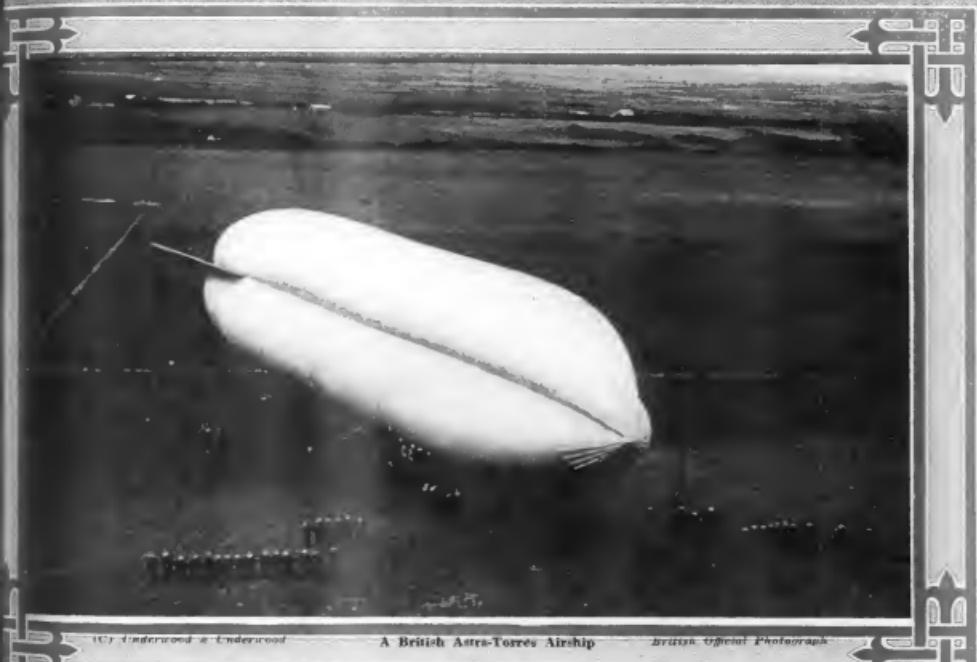


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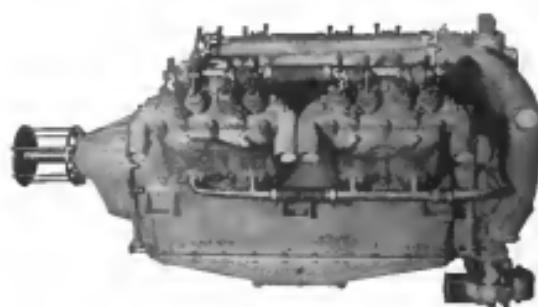
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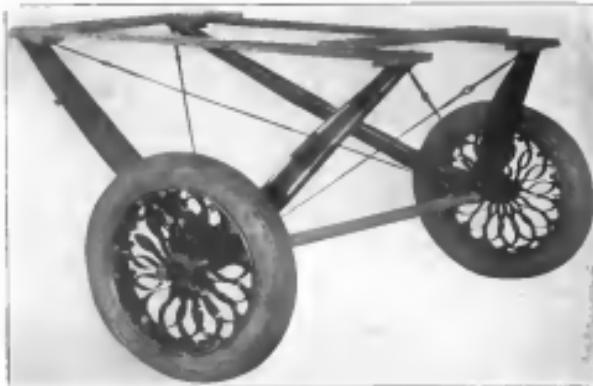

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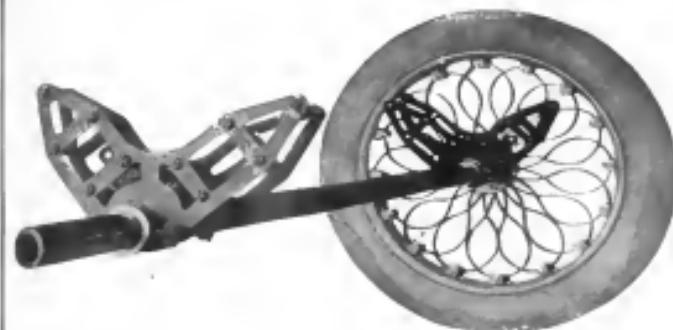
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Vol. 18

November 1, 1927

No. 1

Whirling Table and Method of Testing Model Airscrews at N. P. L.

By A. Page, A.R.C.Sc., D.I.C.

At the present time there are two different methods of testing airscrews at the National Physical Laboratory, (a) with an aircrew dynamometer mounted on a whirling table and, (b) with an aircrew balance designed for use in a wind tunnel.

The present article deals only with the test of these methods

of the whirling table is carried up from the floor to the roof, the arm being supported by connecting the table to a counter-wheel built up of light angle iron and provided with cross bars and steel wire mesh. The whirling table is driven by a 25 h.p. electric motor, through a worm reduction gear of 25 to 1.



Fig. 1. Whirling Table at the N. P. L.

but the author hopes to obtain permission to publish, in his future, a description of the second balance.

Description of the Whirling Table and the Aircrew Dynamometer

A brief description of the whirling table as first designed is given in the "Technical Report of the Advisory Committee on Aeronautics, Year 1920-1921" but since that time both the diameter and the number of aircrew representing have been considerably increased and improved.

The whirling table, a photograph of which is given in Fig. 2, is a universal air shed, 80 ft. by 80 ft., so that the aerofoil experiments may be carried on independently of the atmospheric conditions. The over-all diameter of the whirling table is 80 ft., the dynamometer being mounted at the end of the arm on which is located a pair of light steel tubes having a 15 ft. diameter at the nose to 1 ft. diameter at the extremity. These tubes are spaced 125 ft. apart and are connected together by struts. The central post

is used, serving the central post by the action of the arms when the motor is stopped, the post is not necessarily above the lower wheel and the upper and lower parts connected by a match gear which allows the free rotation of the lower wheel.

The speed of rotation of the whirling table, which is regulated by an adjustable resistance in the armature circuit of the motor, can be varied from 3 to 17 r.p.m., corresponding with theoretical speeds at the end of the arm of 11 to 32 ft. per second.

For the purpose of mounting the aircrew, a seat is mounted on the arm, and also in the mounting apparatus, during the rotation of the whirling table a set of eight slip rings are fixed to the post.

The aerofoil shaft is driven by an endless rubber belt by a 2 h.p. motor mounted on the arm, the speed of the motor being regulated from an observation table in the corner of the whirling table.

A small revolution counter driven by the aerofoil shaft is connected electrically so as to ring a bell every hundred revolu-

*See the later reports of the Advisory Committee for Aeronautics

between, the airspeed being timed with a stop-watch for a period of about two seconds. A second bell rings at each revolution of the whirling table, so that by timing the rings against a stop-watch the speed of rotation of the whirling

table, or air pressure due to the centrifugal load in a de-expanding pipe the reading of the manometer is proportional to $V - v = -F_1$, where $v = \frac{2\pi}{60} R$ from which the velocity of the air swirl may be calculated since F is a measured quantity. The

FIG. 2. PROPELLER DYNAMOMETER—GENERAL ARRANGEMENT

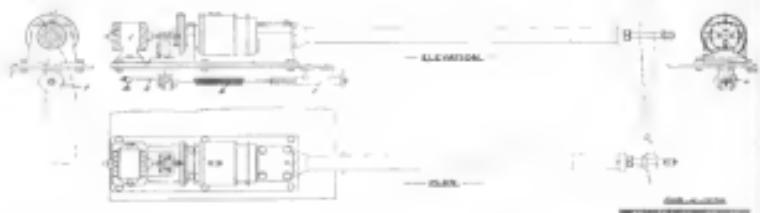


table and hence the speed of translation of the propeller due to motion of the whirling table may be calculated.

As would be expected, the air in the whirling shed is set in motion by the whirling table and also by the locomotion of the aircraft, the velocity of translation of the aircraft through the air being obtained from the measurements of both the velocity of translation of the propeller due to the rotation of the whirling table and the velocity of swirl of the air. The velocity of the air swirl is measured by a Pitot tube mounted at the end of a light arm opposite to that carrying the propeller, the tube following the airstream at a distance of about 100 ft so that it is well out of the region of the direct action of the

speed of the air swirl, when the propeller is not working, about 8 per cent at the max speed.

A measurement of the air swirl is made at each determination of the performance of the aircraft since the swirl is likely to depend on both the rotation speed of the whirling table and the angle of attack of the aircraft in motion. At a low translational speed when the aircraft is moving, the swirl velocity of the air swirl may be negative.

DESCRIPTION OF THE AIRSCREW DYNAMOMETER

The sketches of the airscREW dynamometer, a general arrangement and a sectional drawing, are given in Figs. 3 and 1

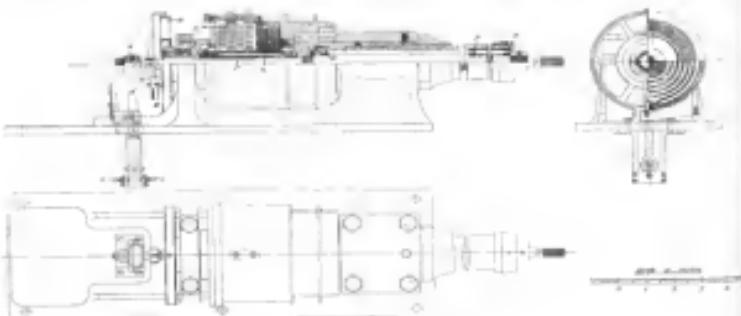


FIG. 3. PROPELLER DYNAMOMETER—SECTION

skip stream. The tube is connected through an airlock manometer and at the top of the central post to one limb of an airscREW, the other limb being open to the atmosphere.

In the case of a propeller at the end of the whirling table, a the velocity of the air swirl in the skip, the velocity of the tube relative to the air is $V - v$, so that, taking into consideration the

skip stream. The dynamometer was designed to measure a maximum load of 15 lb and a maximum torque of 4 lb-in, but with an arrangement of the components greater than 1 lb for the propeller and 1 lb for the dynamometer, the former are only 10 lb.

The propeller shaft is driven through flat coiled springs, the extension of the springs, which is proportional to a

angle, being measured by the relative displacement of a pointer over a small drum n , which makes it avoid slip of the paper. The position of the pointer at any time during the experiment is recorded by passing a series of spaces through the paper; so that if the zero position of the pointer is also

against an oscillating lever d which is pivoted at its center, the lower end of the lever being directly connected by the thrust spring b , the torsion in which is adjusted by the pointer head f , and two adjustable stops g , g' , both stops being oscillated electrostatically from the rest of the dynamometer.

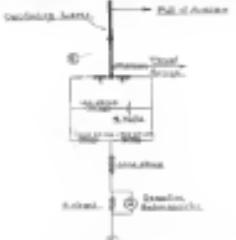


FIG. 4. DIAPHRAGMATED SKETCH OF THRUST BALANCING APPARATUS

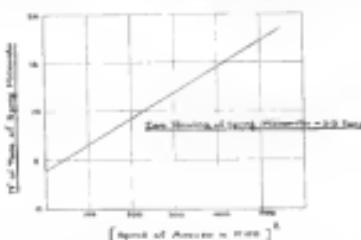
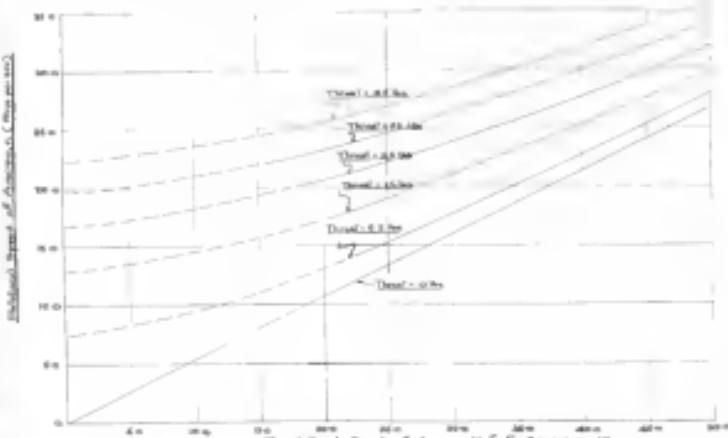


FIG. 5. RELATION OF THRUST TO AIRSPEED AGAINST SQUARE OF ROTATIONAL SPEED OF AIRSCREW

The method of indicating when the thrust of the airscREW is balanced by the puff of the spring is illustrated by the dia phragmated sketch of Fig. 4. From the diagram it is seen that the rotation of the airscREW through the galvanometer depends on the position of the pointer, as shown in Fig. 4. When the thrust of the airscREW balances the puff of the thrust spring, the galvanometer indicates that the puff is at the thrust spring, and the oscillating lever is finding midway between the two stops, i.e., at equal magnitude but of opposite directions through the galvanometer so that the galvanometer needle is not deflected owing to the inertia of the moving parts of the



gives the center of the screw and the radius of the cone.

The sensitivity of the method of balancing is very great, and it will be noted that the sensitivity is not affected by any variation of the battery voltage. Moreover, any variation of the contact resistance at either stop is very small compared with the large resistance of the current. Each stop has a hardened steel post which makes contact with a small flat hardened steel plate at the bottom of the balancing lever.

Overbalance in the balance spring does not call for any loss of the driving torque as the damping conditions by the

governor of known resistance and the whirling added a small amount of resistance. The amount of resistance of the screw is adjusted and the throw of the screw is varied to balance the torque in the thermal spring, the balance being maintained for a period of about two seconds, during which time the translational speed of the end of the whirling arm, the rotational speed of the screws and the velocity of each of the screws are all measured. The moments of inertia of the screws, the center of the screw, and the radii of the screws and translational speeds at the center are being read, the torque is electrically recorded.

The experiments are conducted in two series, one observing the performance of the screws under the thermal balance conditions, the other observing the reduced speed of the screws, the other using the measurements of the relative speed of the arm and the end velocity of the arm. Usually many sets of observations are made in each trials, while the performance of the screws, each set taking about 4 min.

The data of the experiments made with the model screws used in this instance are given in Figs. 5 and 6. In Fig. 5, the relationship between the torque and translational speed at constant values of the torque and torque of Figs. 5 and 6 are to be used for the standard atmosphere of temperature 15.6 deg. C. and pressure 101.3 mm. Hg, the values calculated from the observations of the two series of experiments.

The curves of Figs. 6 and 7, which are plotted directly from experimental observations, are not in a convenient form for a ready calculation of the performance of the screws under working conditions. A more convenient and compact form of presenting the screw data is in terms of absolute coefficients. As the screw data are given in terms of absolute coefficients, the values independent of the nature of the environment in which the screws themselves are dynamically balanced, can, of course, be made available for any desired application.

Absolute coefficients are thus especially suitable for ready numerical computation of experimental data. The results of experiments as calculated from such absolute coefficients are given in the following practical units as will follow the Figs. 5 and 6.

Fig. 4. Rotational speed of the screws
vs. translational speed of the screws
Torsion of screws.

Fig. 5. Relationship between translational speed and torque of screws
Torsion of screws.

Fig. 6. Relationship between translational speed and torque of screws
Torsion of screws.

Fig. 7. Relationship between translational speed and torque of screws
Torsion of screws.

Fig. 8. Relationship between translational speed and torque of screws
Torsion of screws.

coil dash pot E , see Fig. 4. The dash pot connects a series of screws, these alternate discs being wired to a screw carrying the upper ends of the balance springs, and to the screws, to the ends of which are fixed the outer ends of the springs. The vibrations of one coil of a torque spring relative to the other are damped out by the viscosity of the oil between the two screws.

The place of the screw now passes through the axis of rotation of the whirling table, and the whirling force due to the weight of the screws has no component in the direction of the whirling. The overbalanced torque acting on the screws, since it is small compared to the direction of the whirling which is substantially constant, has no effect on the whirling force of a small mass in moving through such a range of motion, Fig. 4.

The Method of Considering the Driving Experiments

It is now proposed to describe the method of performing an accurate experiment, the article being discussed with the data taken from experiments on a model screw. The scale of the model screws was 15.6 mm. the diameter of the balance screws being 9.16 mm.

Firstly, the reading of the peep hole, β , when there is no tension in the spring, is obtained from the statical experiments on the screw, and the corresponding value of T is obtained from the series relating to a fixed point. In Fig. 5 the reading of the peep hole has been plotted against the square of the rotational speed, from which it is seen that for all practical purposes the statical bend of an screw is proportional to the square of the rotational speed.

The reading of the peep hole, when the screws are set rotating and thereby giving no bend, is 5.6 turns, as that from the calibration of the bend spring. It is 5 turns represent 1 lb. the bend of the screws at any reading of the peep hole is known.

The testing of an screw under working conditions similar to those of practice is now considered. The bend spring is

at $T = 5$. In a similar manner we may write either $Q = \frac{1}{2} \pi T^2 D$ or $Q = \frac{1}{2} \pi \frac{D^2}{4} T^2$.

In Fig. 5 the performance of the screws has been expressed in two ways, first, by plotting T and Q , against $(\frac{T}{D})^2$, and second, by plotting T and Q , against $(\frac{Q}{D^2})^2$, the values of $(\frac{T}{D})^2$ and $(\frac{Q}{D^2})^2$ being also at absolute constant. Since $T^2 = T$, $(\frac{T}{D})^2$ being also at absolute constant. Since $T^2 = T$, $(\frac{T}{D})^2$ and $Q^2 = Q$, $(\frac{Q}{D^2})^2$, the screws performance may be

$\frac{1}{2} \pi D^2$ slugs per cu. ft. assuming the temperature of the air to be 15.6 deg. Cent. and the pressure 760 mm.

The value of $(\frac{T}{D})^2$ at which the screw is working is $(\frac{210}{20 \times 9.167})^2$ that is 0.05, the corresponding values of $(\frac{T}{D})^2$ and $(\frac{Q}{D^2})^2$ are measured from the curves being 0.157 and 0.0209 respectively. Hence,

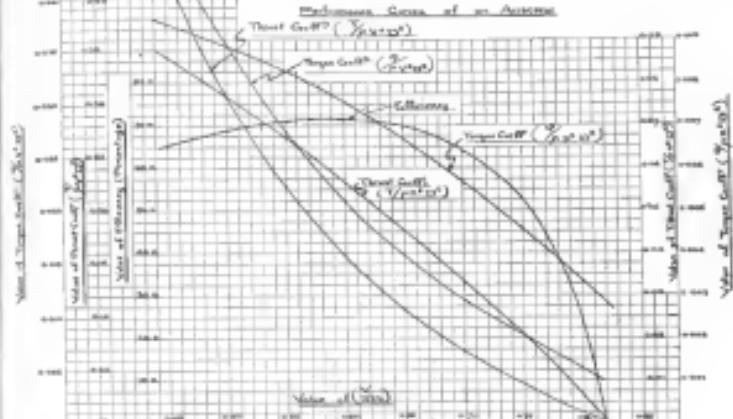


Fig. 5. Relationship between translational speed and torque of screws.

expressed completely by placing either T and Q against $(\frac{T}{D})^2$ or T and Q , against $(\frac{Q}{D^2})^2$.

The difference in the ratio of the useful work done in the screws to the total work per unit the screws is as follows:

$$\begin{aligned} \text{Thread} \times \text{Translational speed} \\ = \frac{T}{2\pi} \times \text{Translational speed} \times \text{Rotational speed} \\ = \frac{T}{2\pi D^2} \times T \times \frac{D}{\pi} = \frac{T^2}{2\pi D^2}. \end{aligned}$$

Efficiency being the ratio of two like quantities, of course, as absolute coefficient. We see, then, that the performance of an screw is to be expressed completely in terms of absolute coefficients as follows:

$$(T, Q, \frac{1}{2} \pi D^2) \text{ or } (T^2, Q^2, \frac{1}{2} \pi D^2).$$

and the usefulness of such a system will be apparent from a consideration of the following.

To calculate the values of the bend and torque of the screws at ground level, when it is moving through the air with a translational speed of 75 m.p.h. and a rotational speed of 2000 rpm., we express the known data in convenient units,

$$\begin{aligned} T = 150 \text{ ft. per sec.} \\ D = 20 \text{ revs. per sec.} \\ \beta = 9.167 \text{ ft.} \end{aligned}$$

$$\begin{aligned} T = 0.00237 \times (9.167)^2 \times (115) > 0.157 = 31.9 \text{ lb.} \\ Q = 0.00237 \times (9.167)^2 \times (318) > 0.0209 = 469 \text{ ft.-lb.} \end{aligned}$$

$$\text{The efficiency } \frac{T^2}{2\pi D^2} = \frac{31.9^2 \times 318 \times 100}{2\pi \times 9.167^2 \times 318} = 72.$$

The thrust and torque could, of course, be calculated from the values of $(\frac{T}{D})^2$ and $(\frac{Q}{D^2})^2$.

$$(\frac{T}{D})^2 \text{ and } (\frac{Q}{D^2})^2.$$

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Landing-Choate
No. 1,220,342.—To William C. Choate, Worcester, Mass. A device for landing aircraft.

1,220,342.—To Joseph L. Choate, Los Angeles, Cal. A airplane.

1,220,342.—To Eugene Paul Hause, Worcester, Mass. Flying apparatus.

1,220,342.—To Louis C. Clegg, Worcester, Eng., assignor of interest to American Motor Car Co., Los Angeles, Calif. Flying apparatus.

1,220,342.—To Maria Estella Charlton, S. D. Strength ability for military purposes.

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Bertie Barnwell, American representative of Whitbread Aircraft, Ltd., Richmond, Surrey, England, recently arrived in America, where plans for establishing with the American airplane manufacturers in the assembly of American machines in the Whitbread works.

Bertie Barnwell and J. A. Whitbread, managing director of the company are native Californians. Mr. Barnwell served in the British Army in France and has been honorably discharged because of wounds and shell shock.

Whitbread Aircraft, Ltd., is one of the largest British manufacturers, is capitalized at over \$6,000,000.

The company has been engaged exclusively in the manufacture of additional plant for enlarged production of flying machines for the British Government.

Airplane Engine Endurance Record

For the first time in the history of the aeronautical testing laboratory at the Wright Field, Dayton, Ohio, an airplane engine has exceeded the German record endurance time. It is considered that engines from all parts of the world have been tested at this laboratory previously, and that not a single one of them has run over 90 hr., the severity of the test is better appreciated.

As in the case of the two previous records, the test engine was not built by an American company, but by a British concern which has until recently been known only in the field of heavy duty marine engines. The experience gained in many years devoted to the design of engines, in which reliability and endurance were of the greatest importance, has evidently had much to do with the success of this engine.

The record was an engine of 340 H.P. which was made by the Beardmore company of London, England. It was established by the same engine which last May was put through a similar test for the German record at Soignies, France, a complete description of which was published in AVIATION, NOVEMBER 1, 1918.

Although details of the test are not now available for publication, it is understood that the engine was run for 100 hr. and stopped after running 24 hr., owing to the blowing out of an oil pump gasket, which, due to consequent loss of pressure, made it impossible to continue the test, although the engine was otherwise in perfect condition and still delivering over its rated 1 power.

The Caproni Machine Flies to Manila

What is reported to be a new record in long distance passenger carrying flight was made Oct. 22 by Lieutenant Rennata, the Italian aviator who flew from Langley Field, Stamp River, Ontario, Canada, to Manila, Philippines, a distance of 3,000 miles over four hours. The Caproni regular biplane had a 200-mile range without a stop. It was powered by a Pobjoy four-cylinder water-cooled engine. A. W. Holt, a Pobjoy agent, drove the car to the meeting point at Langley Field, and Captain Rennata, who had been flying with Holt, took the car to the Pacific Coast. For ten years he was connected with W. H. Grace & Co. of New York.

More Aviator Cadets Wanted

Efforts are being made at the headquarters of the Aviation Department to speed up the rate of enrollment in the Aviation Section of the Regular Officer's Reserve Corps. Twenty thousand men are needed in the man the year 1919 when the War Department intends to send to Europe. Although the Army now better attracts may apply before the examinee himself can be sent, the physical and mental requirements are extremely high and fully 30 per cent of the applicants received are rejected.

An officer in charge of recruiting for the Aviation Section states that candidates are subjected to any severe medical test as has been frequently reported. He says the examination is a man's one, calculated solely to ascertain the applicant's physical and mental fitness for the work of an aviator. Highly physical strength is necessary. In all other respects, however, the examination is one where any man can pass, provided he meets the requirements as to height and weight.

A candidate for commission as flying officer in the Aviation Section must be at least nineteen years old and preferably as much as thirty, although an exception may be made in cases of extreme merit. The man must be in good health, and the lives of others in the saddle, machine, etc., may be at risk. If any of these are not in the examining board of their physical fitness, candidates for commission must have either training or its equivalent. Applicants will not be recommended, who are not in every way qualified and fitted to be recommended after the examination.

All candidates are tested in the Signal Corps or the Regular Officer's Reserve Corps. This service will naturally hold the better the period of training, which covers the entire at the schools of military aeronautics (ground school), and at the aviation or flying school up to the time when, having passed the tests for a general military rating of junior officer and having been promoted by the Signal Officer's Reserve Corps, if they last, they will be discharged from the Signal Corps and revert to their former stations. Failure to qualify for a commission will not exempt from the draft law.

Army Wants Air Mechanics

The Commandant Office of the United States Army at 26 Broad Street, New York City, has made an appeal for mechanics to go to France immediately on presents in the Army Section of the Signal Corps to maintain and repair aircraft.

Major Gen. George O. Squier, Chief Signal Officer of the United States Army, has directed that a recruiting campaign begin at once, with the object of enlisting a total of 100,000 men who will be employed chiefly at the depots of Aeronautics in France. The men will be required to be in good health, either for or for the signal corps or in an aviator, but will remain on horseback at the aviation fields which are occupied by Americans in France. Those who are fit to work in mechanics on the Aviation Service will go to France.

Congeionists, electricians, mechanics, men familiar with gas engines, and others who know how to use their hands are asked to enlist at once.

British Aeronautical Material Agent

It is reported that Arthur F. Chase of A. F. Chase & Co., San Francisco, Calif., has been appointed purchasing agent for the British Government on the Coast for all kinds of aeronautical material and supplies, which will be drawn from that source for aircraft construction and equipment.

In addition to supplies to purchase, Mr. Chase will have charge of all aircraft from Britain that are received on the Pacific Coast.

For ten years he was connected with W. H. Grace & Co. of New York.

New Training Biplane

A new type of training biplane has been flown at Credle Park, N. Y., by Army Instructor Aviation, in which the student sits in front of the pilot's seat. The machine is a biplane, with the upper wing being the one used for training aviators. The two-bay is the product of the Orpheus Engineering Co. and is powered with a 300-hp. Dressemberg engine.



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326 Short, Female	327 Short, Female
326 Long, Female	328 Long, Female
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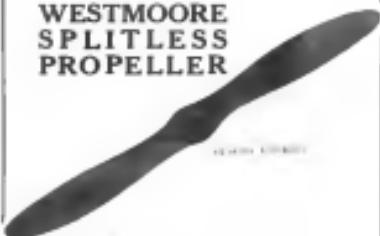
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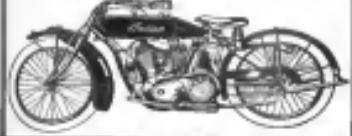
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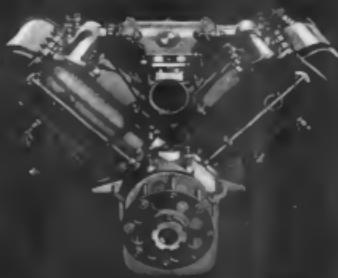


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